### **DECLARATION BY THE CANDIDATE**

Date:\_\_\_\_\_

I hereby declare that the work presented in this dissertation entitled **COMPUTATIONAL STUDY OF TURBINE CASCADE**" has been carried out by me under the guidance of Dr. Samsher, Professor, Department of Mechanical Engineering, Delhi College of Engineering, Delhi and hereby submitted for the partial fulfilment for the award of degree of Master of Engineering in Mechanical Engineering (THERMAL) at Delhi College of Engineering (Delhi university), Delhi.

I further undertake that the work embodied in this major project has not been submitted for the award of any other degree elsewhere.

### CERTIFICATE

It is to certify that the above statement made by the candidate is true to the best of my knowledge and belief.

Dated:-----

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DEVENDRA SINGH

### ABSTRACT

Roughness over the turbine blade is mainly caused by erosion corrosion and deposition. The roughness varies along the blade height and also over different stage of blade. To see the effect of roughness, three profiles have been taken and were checked for different roughness over only pressure surface, only suction surface and over both the surfaces together. The study has been carried out using Fluent software. It has been concluded that the loss coefficient increases with roughness. Roughness over pressure surface is more detrimental than suction surface. Study also reveals that roughness over reaction profile is more detrimental than that of impulse profile.

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Coordinates of the profile 3525 of Samsher [1]

Coordinates of the profile 6030 of Samsher [1]

Coordinate of the profile 5530 of Samsher [1]

## NOMENCLATURE

## Nomenclature

E	Both suction and pressure surface of blade rough
PE	Entire pressure surface of blade rough
PL	1/3 <sup>rd</sup> of pressure surface of blade rough at leading edge
PM	1/3 <sup>rd</sup> of pressure surface of blade rough at mid chord
РТ	1/3 <sup>rd</sup> of pressure surface of blade rough at trailing edge
RKE	Realizable k-ɛ model
RNG	Renormalized group model
S	Both suction and pressure surface of blade smooth
SE	Entire suction surface of blade rough
SL	1/3 <sup>rd</sup> of suction surface of blade rough at leading edge
SM	1/3 <sup>rd</sup> of suction surface of blade rough at mid chord

# Symbols

а	Local velocity of sound
С	Chord length of blade
$C_p$	Specific heat at constant pressure
D <sub>H</sub>	Hydraulic diameter
E	Energy of the system
Fi	External body force in 'i' direction
$g_i$	Acceleration due to gravity in i direction

k <sub>s</sub>	Equivalent sand grain roughness
K <sub>adm</sub>	Admissible surface roughness
I	Length /span of blade
L	Characteristic length
m	Mass flow of the system
Μ	Mach number
Р	Pressure of system
R <sub>e</sub>	Reynolds number
S	Pitch
S <sub>m</sub>	Source term
т	Temperature of the system
<i>u</i> <sub>i</sub>	Velocity component in i direction
$\overline{u_i}$	Average velocity in turbulent flow in i direction
V	Velocity of fluid
C <sub>x1</sub>	Inlet velocity
Cx <sub>2</sub>	outlet velocity
α <sub>1</sub>	Inlet angle (air angle)
α2	Exit angle, Blade angle

# **Greek symbols**

ρ	Density of air
μ	Dynamic viscosity
К	Turbulent kinetic energy
Е	Rate of dissipation of turbulent kinetic energy
γ	Ratio of specific heats
$\phi$	Scalar variable
η	Efficiency
$\sigma_{\scriptscriptstyle k}$	Prandtl number for turbulent kinetic energy
$\sigma_{_{arepsilon}}$	Prandtl number for dissipation

- Wall shear stress  $\tau_{\omega}$
- ζ Mass averaged value of profile loss coefficient
- Local profile loss coefficient  $\zeta_y$
- Local exit flow angle  $\beta_{2y}$
- $\delta_{_{1k}}$ Boundary layer thickness
- $\delta_{ii}$ Kronecker's delta
- $au_{ij}$ Stress tensor

### Subscripts

y

- 1 Inlet to cascade 2 Outlet to cascade Total value 0 Static value
- S Local value in pitch-wise direction